

## Flow Across Cylinder and Spheres

### A) Cylinder

#### 1) Circular Cylinder

The most comprehensive relation for the flow across circular cylinder in which applicable over the complete range of available data

$$\text{Nu}_d = 0.3 + \frac{0.62 \text{Re}_d^{1/2} \text{Pr}^{1/3}}{[1 + (0.4/\text{Pr})^{2/3}]^{1/4}} \left[ 1 + \left( \frac{\text{Re}_d}{282,000} \right)^{5/8} \right]^{4/5}$$

for  $10^2 < \text{Re}_d < 10^7$ ;  $\text{Pe}_d > 0.2$

For more accurate value of  $\text{Nu}_d$  in the midrange of Reynolds numbers between 20,000 and 400,000, it is recommended to use the following relation

$$\text{Nu}_d = 0.3 + \frac{0.62 \text{Re}_d^{1/2} \text{Pr}^{1/3}}{[1 + (0.4/\text{Pr})^{2/3}]^{1/4}} \left[ 1 + \left( \frac{\text{Re}_d}{282,000} \right)^{1/2} \right]$$

for  $20,000 < \text{Re}_d < 400,000$ ;  $\text{Pe}_d > 0.2$

#### 2) Non-Circular Cylinder

For the flow across non-circular cylinder, the following equation may be used

$$\text{Nu}_{df} = \frac{hd}{k_f} = C \left( \frac{u_\infty d}{v_f} \right)^n \text{Pr}_f^{1/3}$$

Where the constants  $C$  and  $n$  are tabulated in Table 6-2 below. Properties for using this equation are evaluated at the film temperature as indicated by the subscript  $f$ .

**Table 6-2** | Constants for use with Equation (6-17), based on References 8 and 9.

$\text{Re}_{df}$	$C$	$n$
0.4–4	0.989	0.330
4–40	0.911	0.385
40–4000	0.683	0.466
4000–40,000	0.193	0.618
40,000–400,000	0.0266	0.805

### B) Sphere

For the heat transfer from spheres to a flowing gas across it, the following relation may be used:

$$\frac{hd}{k_f} = 0.37 \left( \frac{u_\infty d}{\nu_f} \right)^{0.6} \quad \text{for } 17 < \text{Re}_d < 70,000$$

A more important relation is a single equation for gases and liquids flowing across spheres:

$$\text{Nu} = 2 + (0.4 \text{Re}_d^{1/2} + 0.06 \text{Re}_d^{2/3}) \text{Pr}^{0.4} (\mu_\infty / \mu_w)^{1/4}$$

Which is valid for the range  $3.5 < \text{Re}_d < 8 \times 10^4$  and  $0.7 < \text{Pr} < 380$ . Properties in this equation are evaluated at the free-stream temperature  $T_\infty$ .

### EXAMPLE 6-7

### Airflow Across Isothermal Cylinder

Air at 1 atm and 35°C flows across a 5.0-cm-diameter cylinder at a velocity of 50 m/s. The cylinder surface is maintained at a temperature of 150°C. Calculate the heat loss per unit length of the cylinder.

#### ■ Solution

We first determine the Reynolds number and then find the applicable constants from Table 6-2 for use with Equation (6-17). The properties of air are evaluated at the film temperature:

$$T_f = \frac{T_w + T_\infty}{2} = \frac{150 + 35}{2} = 92.5^\circ\text{C} = 365.5 \text{ K}$$

$$\rho_f = \frac{p}{RT} = \frac{1.0132 \times 10^5}{(287)(365.5)} = 0.966 \text{ kg/m}^3 \quad [0.0603 \text{ lb}_m/\text{ft}^3]$$

$$\mu_f = 2.14 \times 10^{-5} \text{ kg/m} \cdot \text{s} \quad [0.0486 \text{ lb}_m/\text{h} \cdot \text{ft}]$$

$$k_f = 0.0312 \text{ W/m} \cdot ^\circ\text{C} \quad [0.018 \text{ Btu/h} \cdot \text{ft} \cdot ^\circ\text{F}]$$

$$\text{Pr}_f = 0.695$$

$$\text{Re}_f = \frac{\rho u_\infty d}{\mu} = \frac{(0.966)(50)(0.05)}{2.14 \times 10^{-5}} = 1.129 \times 10^5$$

From Table 6-2

$$C = 0.0266 \quad n = 0.805$$

so from Equation (6-17)

$$\frac{hd}{k_f} = (0.0266)(1.129 \times 10^5)^{0.805} (0.695)^{1/3} = 275.1$$

$$h = \frac{(275.1)(0.0312)}{0.05} = 171.7 \text{ W/m}^2 \cdot ^\circ\text{C} \quad [30.2 \text{ Btu/h} \cdot \text{ft}^2 \cdot ^\circ\text{F}]$$

The heat transfer per unit length is therefore

$$\begin{aligned} \frac{q}{L} &= h\pi d(T_w - T_\infty) \\ &= (171.7)\pi(0.05)(150 - 35) \\ &= 3100 \text{ W/m} \quad [3226 \text{ Btu/ft}] \end{aligned}$$

## Heat Transfer from Electrically Heated Wire

## EXAMPLE 6-8

A fine wire having a diameter of  $3.94 \times 10^{-5}$  m is placed in a 1-atm airstream at  $25^\circ\text{C}$  having a flow velocity of 50 m/s perpendicular to the wire. An electric current is passed through the wire, raising its surface temperature to  $50^\circ\text{C}$ . Calculate the heat loss per unit length.

■ **Solution**

We first obtain the properties at the film temperature:

$$\begin{aligned} T_f &= (25 + 50)/2 = 37.5^\circ\text{C} = 310 \text{ K} \\ v_f &= 16.7 \times 10^{-6} \text{ m}^2/\text{s} \quad k = 0.02704 \text{ W/m} \cdot ^\circ\text{C} \\ \text{Pr}_f &= 0.706 \end{aligned}$$

The Reynolds number is

$$\text{Re}_d = \frac{u_\infty d}{v_f} = \frac{(50)(3.94 \times 10^{-5})}{16.7 \times 10^{-6}} = 118$$

The Peclet number is  $\text{Pe} = \text{Re} \text{Pr} = 83.3$ , and we find that Equations (6-17), (6-21), or (6-19) apply. Let us make the calculation with both the simplest expression, (6-17), and the most complex, (6-21), and compare results.

Using Equation (6-17) with  $C = 0.683$  and  $n = 0.466$ , we have

$$\text{Nu}_d = (0.683)(118)^{0.466}(0.705)^{1/3} = 5.615$$

and the value of the heat-transfer coefficient is

$$h = \text{Nu}_d \left( \frac{k}{d} \right) = 5.615 \frac{0.02704}{3.94 \times 10^{-5}} = 3854 \text{ W/m}^2 \cdot ^\circ\text{C}$$

The heat transfer per unit length is then

$$\begin{aligned} q/L &= \pi dh(T_w - T_\infty) = \pi(3.94 \times 10^{-5})(3854)(50 - 25) \\ &= 11.93 \text{ W/m} \end{aligned}$$

Using Equation (6-21), we calculate the Nusselt number as

$$\begin{aligned} \text{Nu}_d &= 0.3 + \frac{(0.62)(118)^{1/2}(0.705)^{1/3}}{[1 + (0.4/0.705)^{2/3}]^{1/4}} [1 + (118/282,000)^{5/8}]^{4/5} \\ &= 5.593 \end{aligned}$$

and

$$h = \frac{(5.593)(0.02704)}{3.94 \times 10^{-5}} = 3838 \text{ W/m}^2 \cdot ^\circ\text{C}$$

and

$$q/L = (3838)\pi(3.94 \times 10^{-5})(50 - 25) = 11.88 \text{ W/m}$$

**EXAMPLE 6-9****Heat Transfer from Sphere**

Air at 1 atm and 27°C blows across a 12-mm-diameter sphere at a free-stream velocity of 4 m/s. A small heater inside the sphere maintains the surface temperature at 77°C. Calculate the heat lost by the sphere.

**■ Solution**

Consulting Equation (6-30) we find that the Reynolds number is evaluated at the free-stream temperature. We therefore need the following properties: at  $T_\infty = 27^\circ\text{C} = 300\text{ K}$ ,

$$\nu = 15.69 \times 10^{-6} \text{ m}^2/\text{s} \quad k = 0.02624 \text{ W/m} \cdot ^\circ\text{C},$$

$$\text{Pr} = 0.708 \quad \mu_\infty = 1.8462 \times 10^{-5} \text{ kg/m} \cdot \text{s}$$

At  $T_w = 77^\circ\text{C} = 350\text{ K}$ ,

$$\mu_w = 2.075 \times 10^{-5}$$

The Reynolds number is thus

$$\text{Re}_d = \frac{(4)(0.012)}{15.69 \times 10^{-6}} = 3059$$

From Equation (6-30),

$$\begin{aligned} \overline{\text{Nu}} &= 2 + [(0.4)(3059)^{1/2} + (0.06)(3059)^{2/3}](0.708)^{0.4} \left( \frac{1.8462}{2.075} \right)^{1/4} \\ &= 31.40 \end{aligned}$$