

Spring –damper system

1. The force exerted by the spring (toward the left) of $-ky$ where k is a positive constant, called Hooke's constant.
2. The viscous friction force (acting to the left) of $-c \, dy/dt$, where c is positive constant called the damping coefficient.
3. The external force $F(t)$ (acting toward the right)

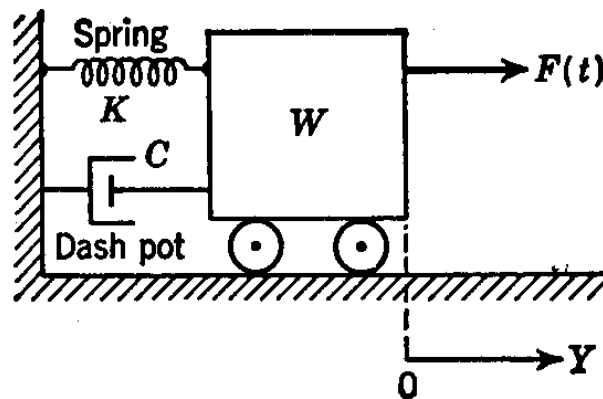


Fig.26 Spring –damper system

Force balance :

$$F(t) - F_s - F_d = (\text{mass} \times \text{acceleration})$$

$$F(t) - ky(t) - \frac{c \, dy(t)}{dt} = m \frac{d^2 y}{dt^2}$$

$$m \frac{d^2 y}{dt^2} + \frac{c \, dy(t)}{dt} + ky = F(t)$$

$$\frac{m}{k} \frac{d^2 y}{dt^2} + \frac{c}{k} \frac{dy(t)}{dt} + y(t) = \frac{1}{k} F(t)$$

$$\tau = \sqrt{\frac{m}{k}} \text{ (sec) } , \zeta = \frac{1}{2} \frac{c}{k} \sqrt{\frac{k}{m}} = \frac{c}{2} \sqrt{\frac{1}{mk}}$$

$$G(s) = \frac{y(s)}{F(s)} = \frac{1/k}{\tau^2 s^2 + 2\zeta\tau s + 1}$$

$$\frac{y(s)}{x(s)} = \frac{1}{\tau^2 s^2 + 2\zeta\tau s + 1}$$

$$s_1, s_2 = \frac{-2\zeta\tau \pm \sqrt{(2\zeta\tau)^2 - 4\tau^2}}{2\tau^2}$$

$$s_1, s_2 = \frac{-\zeta}{\tau} \pm \frac{\sqrt{\zeta^2 - 1}}{\tau}$$

Analysis of Second order system

$$\tau^2 = \frac{1}{\omega_n^2}$$

$$\omega_n = 2\pi f_n$$

ω_n : natural frequency rad/s

f_n : cyclical freq. (Hz)

ζ =damping factor

if a unit step in $x(t)$, $x(s) = \frac{1}{s}$

$$y(s) = \frac{k}{s(\tau^2 s^2 + 2\zeta\tau s + 1)}$$

The characteristic eq. $\tau^2 s^2 + 2\zeta\tau s + 1 = 0$

$$s_1, s_2 = \frac{-2\zeta\tau \pm \sqrt{(2\zeta\tau)^2 - 4\tau^2}}{2\tau^2}$$

$$s_1, s_2 = \frac{-\zeta}{\tau} \pm \frac{\sqrt{\zeta^2 - 1}}{\tau}$$

1. If $\zeta > 1$:unequal root ,-ve and real roots $= -m_1, m_2$

$$y(t) = e^{-m_1 t} + e^{-m_2 t}$$

The system is not oscillatory and over damped

2. If $\zeta = 1$: equal root ,-ve and real roots $= -m$

$$y(t) = (At + B)e^{-tm}$$

The system is critically damped response

3. $0 < \zeta < 1$, complex and -ve real part

$$s_1 \text{ and } s_2 = -\alpha \pm j\beta$$

$$y(t) = e^{-\alpha t} \sin \beta t$$

The system is under damped response

4. $\zeta = 0$, pure imaginary

$$s_1 \text{ and } s_2 = \pm j\beta$$

$$y(t) = \sin \beta t$$

The system is continuous oscillation

5. $\zeta < 0$, complex and +ve real part

$$s_1 \text{ and } s_2 = \alpha \pm j\beta$$

$$y(t) = e^{\alpha t} \sin \beta t$$

The system is oscillation and unstable

In general for stable system the optimum of $\zeta = 0.214$